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Possibilities of Calculation of Convection Heat Transfer inside Built-up Urban Area

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ABSTRACT

Article History Accurate modelling of convective heat transfer processes is imperative for the assessment of Article # 24-1004 the urban heat island effect and the development of measures to enhance the urban Received: 02-Dec-24 microclimate. Existing studies predominantly rely on field observations and wind tunnel Revised: 24-Dec-24 experiments, while hydrodynamic modelling endeavors often fall short in providing an Accepted: 29-Dec-24 Online First: 09-Jan-25 accurate representation of heat distribution. This paper proposes a solution grounded in the Navier-Stokes equation, employing a synthesis of an evolutionary approach and multi-variable optimization. The GAMS programming language was utilized to facilitate the calculation. The efficacy of the proposed approach was substantiated through the verification of its performance on a test problem pertaining to the occurrence of convective water motion. This demonstrated the approach's capacity to accurately fulfil the water balance. Subsequent to this, the proposed approach was employed to address the convective heat transfer issue in an urban area, leading to the formulation of a model that delineated the temperature distribution around buildings. This model successfully ensured the precise compliance with the laws of conservation of mass and the absence of wave phenomena.

Keywords: Microclimate, Urban heat island, Convective heat transfer, Heat and mass retention, Navier-Stokes equations, Air flow motion.

INTRODUCTION

The urban heat island effect, or UHI, is a well-documented phenomenon that has been extensively researched in numerous cities worldwide (Yuan & Bauer, 2007; Blocken et al., 2011; Gagliano et al., 2014; Santamouris, 2014; Minh & Nguyen, 2019). It is characterized by a rise in temperature ranging from 2 to 5°C within urban areas compared to the surrounding rural regions, persisting for several hours daily (Memon et al., 2008; Minh & Nguyen, 2019). Oke (1982) and Luo et al. (2020) observed a temperature disparity of 5 to 10°C, with some studies documenting temperatures as high as 8 to 15°C.

Urban heat islands (UHI) are an increasingly problematic phenomenon in modern cities, particularly in the context of global climate change. In particular, there is an increased load on urban electricity systems due to the increased consumption of electricity by air conditioning (Turhan et al., 2023; Panwar & Jindal, 2024). Consequently, numerous researchers are

investigating the causes and processes of urban heat island formation.

The formation of UHI is influenced by various factors, including artificial temperature and humidity, pollutants, and impervious surfaces, as well as thermal radiation in buildings. This subject has been explored by Minh & Nguyen (2019), Minh & Shukurov (2020) and Ismael et al. (2024). Additionally, the relationship between high temperatures on urban surfaces and the high albedo percentages of road and building surfaces has been investigated by several researchers including Yuan & Bauer (2007), Gago et al. (2013), Gagliano et al. (2014), Santamouris (2014), Minh & Nguyen (2019), Awol et al. (2020), and Mehrotra et al. (2020). These studies also examined the relationship between the heat island effect and the basis of urban design, as well as the potential for mitigating the UHI effect through the reflectivity of buildings. The shape (Fan et al., 2019) and orientation (Mohajer et al., 2023) of buildings relative to the sun and to the direction of prevailing winds is also important.

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The formation of UHI is influenced by various factors, transportation. includina urban motorized The uninterrupted flow of vehicles results in elevated levels of heat and pollutant emissions. In metropolitan areas characterized by substantial vehicular traffic, the level of heat emitted is notably higher compared to areas with a limited number of automobiles. Studies conducted in major metropolitan areas such as Chicago and New York have demonstrated a positive correlation between concentrated traffic flows and the intensity of UHI. Public transportation, encompassing buses, trolleybuses, and trams, contributes to UHI formation, though to a lesser extent than passenger cars. Conversely, transportation modes such as subways have been shown to reduce overall heat generation by reducing car traffic (Haddad et al., 2015). The development of infrastructure designed for pedestrians and cyclists has been shown to reduce the dependency on private vehicles, thereby contributing to a decrease in the UHI effect. The findings of several studies indicate a correlation between the development of sustainable transport infrastructure and a reduction in the magnitude of urban heat island effects. Mitigation of the UHI effect is increasingly being pursued through the promotion of sustainable urban transport and green infrastructure (Irfeey et al., 2023).

The predominant causes of the heat island effect are generally recognized to be the intricate geometry of streets in the presence of high-rise buildings, which hinders the absorption of solar radiation by building elements and impedes heat exchange; the thermal properties of building and road surfaces enhance heat retention within the metropolis.

The "greenhouse effect" in urban areas is more pronounced due to the presence of a more polluted atmosphere. This effect is expressed more strongly than in rural areas. The effective albedo of the urban system is reduced due to multiple reflection of solar radiation by the surfaces of buildings. This is also caused by the absorption of heat by the walls of buildings. Additionally, evaporative cooling surfaces are reduced by the soil surface or plant crowns. The aforementioned factors lead to uneven heating of the surface and local convective movement of air masses in urban areas. Consequently, these urban areas experience the formation of specific air mass flows, which are often referred to as heat island effects. These flows can either cool heat islands within cities or, at times, lead to an increase in temperatures within these heat islands. However, it is crucial to acknowledge that this analysis merely documents the existing reality and does not offer a comprehensive solution to address the underlying issues (Yuan & Bauer, 2007; Blocken et al., 2011; Minh & Nguyen, 2019).

In such instances, endeavors to enhance the circumstances are comprised of measures that are subjected to recurrent evaluation through a series of observations. The pursuit of solutions is predominantly executed through the reiteration of measures conducted in disparate locations. However, it is evident that each urban heat island possesses distinct characteristics, rendering such an approach to be ineffective in all instances. These

methodologies yield a comprehensive overview of the prevailing temperature distribution. Nevertheless, to exert a substantial influence on heat transfer processes within urban environments, mathematical modeling is imperative (Jia et al., 2025). The calculation of thermal convection inside a high-rise urban zone will allow for more accurate determination of the most significant causes of heat islands, enabling the optimization of efforts and resources prevent excessive heating of high-rise building to groups. The efficacy of these measures will be enhanced in terms of both cost and impact. It is important to note that comparisons between heat islands within a single city are only of an evaluative nature, and measures implemented to combat a heat island in one area cannot be directly transferred to another heat island, even within the same city. This is due to the presence of a unique structure of convective transfer for each group of buildings in each area (Nekkanti & Radhakrishnan, 2024).

According to Nottrott et al. (2011), the mechanism of heat transfer in the convective near-wall boundary layer on the leeward vertical wall of a building was studied experimentally. Studies show that the leeward walls of buildings in urban environments are dominated by a regime of turbulent natural convection. This has important implications for the parameterization of convective heat fluxes in urban areas. Building density has a significant influence on convective heat transfer coefficients. In isolated flows, these coefficients change dramatically with increasing density, whereas in the interaction and slip regimes they decrease smoothly with increasing building density (Liu et al., 2015; Awol et al., 2020). The only way to predict thermal convection within groups of buildings is to solve hydrodynamic problems (Blocken et al., 2011; Gagliano et al., 2014).

Computational fluid dynamics (CFD) modeling is frequently employed as a means of devising efficacious strategies for the mitigation of thermal effects. In urban environments, the convective transport of heat is influenced by a variety of factors, including the geometry of buildings, the physical properties of materials, and the prevailing climatic conditions (Donthu et al., 2024). The growing reliance on CFD modeling can be attributed to its ability to visualize and predict the distribution of airflow and temperature in urban areas. A range of software packages, such as ANSYS Fluent and OpenFOAM, are utilized to facilitate CFD modeling, thereby enabling the detailed modeling of both outdoor and indoor environments. In their work, Mirsadeghi et al. (2013) demonstrated the use of these tools to estimate the exterior heat transfer coefficients required for the energyefficient design of buildings. Convective transport in urban areas has been a subject of study for numerous researchers. For instance, Moediartianto et al. (2018) conducted field and wind tunnel studies, in addition to three-dimensional Reynolds-averaged Navier-Stokes equation analyses, to study the convective heat transfer coefficient of urban roofs (Blocken et al., 2011; Minh & Nguyen, 2019). However, it was found that the accuracy of the results obtained was insufficient for accurate modeling of the urban climate.

The advent of a highly efficient, conservative finite difference scheme for the transfer of matter and thermal energy was not yet in existence. The scheme, which allows the laws of conservation of matter and energy to be accurately followed, was first used in the problem described in this article. The versatility of these tools in modeling different urban configurations renders them invaluable for predicting the impact of architectural decisions on urban thermodynamics. The application of CFD modeling to investigate heat transfer in urban buildings depending on various factors has been considered by numerous authors, including Chung & Choo (2011), Mirsadeghi et al. (2013), and Schrijvers et al. (2018). While CFD offers numerous advantages, including comprehensive insight into airflow and temperature interactions, it is not without its limitations. The reliability of CFD results is contingent upon the quality of the input data and the validation of the model. As Jun Chung and Choo (2011) observe, discrepancies in the input parameters can result in considerable discrepancies in the outcomes.

Researchers have noted the complexity of hydrodynamic modeling of convective heat transfer processes in urban buildings and the lack of sufficiently accurate results (Yuan & Bauer, 2007; Blocken et al., 2011; Gagliano et al., 2014; Minh & Nguyen, 2019). This observation suggests that a universally accepted solution to this problem remains elusive.

The objective of this study is to demonstrate the feasibility and existence of a mathematical model that captures the dynamics of convective heat transfer within urban buildings. This model is formulated using the equations of hydrodynamics, thereby ensuring full compliance with the laws of conservation of moving masses, energy, and heat. The efficiency and simplicity of the method of describing obstacles in the path of air mass movement by means of the interaction of the air flow with a certain virtual additional phase with certain special properties is shown. This solution can serve as a foundation for modeling the microclimate of urban buildings, thereby substantiating urban planning initiatives and establishing zones of climatic comfort.

MATERIALS & METHODS

The calculation of heat transfer by convection is based on the system of Navier-Stokes equations "truncated" to hydrostatics and with the assumption of incompressibility of the moving medium (Marchuk et al., 1975; Marchuk & Kurbatkin, 1978; Marchuk, 1980; Batchelor & Moffat, 1984; Anderson et al., 1990). It is imperative to acknowledge that this "truncation" is implemented to mitigate the occurrence of sound and shock waves. Absent this truncation, numerical analogues of existing shock and sound waves are inevitable during the computational process. The emergence of shock wave analogues can be attributed to rounding errors, the order of approximation by finite-difference analogues of differential operators, and inconsistency during the establishment of initial conditions. The foundation for subsequent calculations is the system of Navier-Stokes equations (1) reduced to hydrostatics (Furukawa et al., 2020; Gao et al., 2021). This

system is widely employed in calculations of air and water circulation (Lax & Richtmyer, 1956; Marchuk et al., 1975; Marchuk & Kurbatkin, 1978; Marchuk, 1980).

$$\frac{\partial V_x}{\partial t} + V_x \frac{\partial V_x}{\partial x} + V_y \frac{\partial V_x}{\partial y} + V_z \frac{\partial V_x}{\partial z} = -\frac{1}{\varrho} \cdot \frac{\partial P}{\partial x} + \nu \cdot \Delta V_x$$

$$\frac{\partial V_y}{\partial t} + V_x \frac{\partial V_y}{\partial x} + V_y \frac{\partial V_y}{\partial y} + V_z \frac{\partial V_y}{\partial z} = -\frac{1}{\varrho} \cdot \frac{\partial P}{\partial y} + \nu \cdot \Delta V_y$$

$$P = g \cdot \int_z^{Z_P(x,y)} \varrho \cdot \partial z$$

$$\frac{\partial V_x}{\partial x} + \frac{\partial V_y}{\partial y} + \frac{\partial V_z}{\partial z} \approx 0$$
(1)

Where: $V_{x_t} V_{y_t} V_z$ are components of the velocity vector along the coordinate axes OX, OY, OZ,

P - pressure,

ν – kinematic viscosity,

Zp(x, y) – position of some surface in the calculation of gas motion or position of the free surface of the liquid in the calculation of liquid motion,

z - vertical coordinate,

q - free-fall acceleration,

 $\Delta = \frac{\partial^2}{(\partial x)^2} + \frac{\partial^2}{(\partial y)^2} + \frac{\partial^2}{(\partial z)^2} -$ the Laplacean differential operator.

Density in such systems is assumed to be weakly variable and that is why we use the approximate equality sign in the incompressibility equation of the medium.

Let us add to the system (1) the equation of conservation of thermal energy (2) (Wei et al., 2021; Ricardo et al., 2024):

$$\frac{\partial T}{\partial t} + V_x \cdot \frac{\partial T}{\partial x} + V_y \cdot \frac{\partial T}{\partial y} + V_z \cdot \frac{\partial T}{\partial z} = \frac{\lambda}{C \cdot \varrho} \cdot \bigtriangleup T$$
(2)
where: T is the temperature,
$$\lambda - \text{heat transfer coefficient,}$$

C – specific heat capacity.

Adding to the system of equations (1) and (2) the equation of state (Marchuk et al., 1975; Roache, 1976; Marchuk & Kurbatkin, 1978; Marchuk, 1980), we close the system with respect to the variables $V_{x_t} V_{y_t} V_{z_t} P_t T_t \rho$ $\varrho = \Phi(T)$

(3) Where: $\Phi(T)$ is a regularity linking the density of the medium and its temperature.

In hydrodynamic calculations, when preparing an approximate solution algorithm, the incompressibility equation (4) is integrated vertically and the obvious fact that in the vertical direction the velocity of the free surface coincides with the vertical velocity of particles lying on this surface is used. The result of such actions is the appearance of the evolutionary equation for determining the position of the free surface (5). Evolutionary equations with respect to some variable are equations in which the first time derivative of this variable is present.

$$\int_{Z_{g}}^{Z_{p}} \left(\frac{\partial V_{x}}{\partial x} + \frac{\partial V_{y}}{\partial y} \right) \partial z = -\int_{Z_{g}}^{Z_{p}} \frac{\partial V_{z}}{\partial z} \partial z = -V_{z}|_{z=Z_{p}}$$

$$\frac{\partial Z_{p}}{\partial t} + V_{x} \cdot \frac{\partial Z_{p}}{\partial x} + V_{y} \cdot \frac{\partial Z_{p}}{\partial y} = V_{z}|_{z=Z_{p}}$$

$$(4)$$

Let us substitute into the horizontal components of the equation of motion the hydrostatic pressure:

$$P = g \cdot \int_{z}^{z_{P}(x,y)} \varrho \cdot \partial z$$
(6)

By application of the Leibniz formula for the differentiation of a definite integral with variable integration limits [8], we obtain system (7):

$$\begin{cases} \frac{\partial V_x}{\partial t} + V_x \frac{\partial V_x}{\partial x} + V_y \frac{\partial V_x}{\partial y} + V_z \frac{\partial V_z}{\partial z} = -\frac{g}{\varrho} \cdot \left(\int_z^{2p(xy)} \frac{\partial \varrho(x, z)}{\partial x} \cdot \partial z + \frac{\partial Zp(x, y)}{\partial x} \cdot \varrho(x, y, z)|_{z=Zp(x, y)} \right) + v \cdot \Delta V_x \\ \frac{\partial V_y}{\partial t} + V_x \frac{\partial V_y}{\partial x} + V_y \frac{\partial V_y}{\partial y} + V_z \frac{\partial V_y}{\partial z} = -\frac{g}{\varrho} \cdot \left(\int_z^{2p(xy)} \frac{\partial (z, y, z)}{\partial y} \cdot \partial z + \frac{\partial Zp(x, y)}{\partial y} \cdot \varrho(x, y, z)|_{z=Zp(x, y)} \right) + v \cdot \Delta V_y \\ \frac{\partial Z_p}{\partial t} + V_x \cdot \frac{\partial Z_p}{\partial x} + V_y \cdot \frac{\partial Z_p}{\partial y} = V_z|_{z=Z_p} = -\int_{Z_g}^{Z_p} \left(\frac{\partial V_x}{\partial x} + \frac{\partial V_y}{\partial y} \right) \partial z \\ \frac{\partial T}{\partial t} + V_x \cdot \frac{\partial T}{\partial x} + V_y \cdot \frac{\partial T}{\partial y} + V_z \cdot \frac{\partial T}{\partial z} = a \cdot (\Delta T) \\ \varrho = \Phi(T) \end{cases}$$

The evolution equation for determining the surface Z_p allows us to solve the system (7) by the establishment method, applying easy and comprehensible calculation algorithms and a well-developed tool of finite-difference schemes. However, when solving the system of equations (7), there are also difficulties that are difficult to eliminate.

Considering together the first, second and third equations of the system (7), we note that the group of these three equations describes the propagation of waves on the surface Z_p . The velocity of wave motion on the surface Z_p will be equal to the square root of the product of the acceleration of free fall by the depth of the flow of the moving medium under the surface Z_p. This velocity will be the main constraint for choosing the maximum possible time step. Convective motions of the medium slower than the wave velocity at the Zp surface will be strongly smoothed by the action of the scheme viscosity. The consequence will be low informativeness of the obtained approximate solutions. Specialists working in the field of computational hydromechanics apply artificial heuristic digital constructions in the task of which is the destruction or reduction of high-speed waves on the surface Z_p preserving the possibility of smooth changes in the position of the surface Z_p .

A slightly different method of destroying surface waves is proposed. This method is not artificial. The method is based on the obvious fact that in a stable and unchanging solution, the surface must be stable as well.

That is, the value of $V_z|_{z=Z_p}$ must be equal to zero. Therefore, instead of a closed system of equations (7), assuming the presence of a single solution, it is proposed to solve a non-closed optimisation problem in which from the set of surface locations the one at which the value of $V_z|_{z=Z_p}$ will be minimal will be chosen. That is, it is proposed to solve the system of equations (8):

$$\begin{cases} \frac{\partial V_x}{\partial t} + V_x \frac{\partial V_x}{\partial x} + V_y \frac{\partial V_x}{\partial y} + V_z \frac{\partial V_z}{\partial z} = -\frac{g}{\varrho} \cdot \left(\int_{z}^{zp(x,y)} \frac{\partial \varrho(x,z)}{\partial x} \cdot \partial z + \frac{\partial Zp(x,y)}{\partial x} \cdot \varrho(x,y,z) |_{z=Zp(x,y)} \right) + \nu \cdot \Delta V_x \\ \frac{\partial V_y}{\partial t} + V_x \frac{\partial V_y}{\partial x} + V_y \frac{\partial V_y}{\partial y} + V_z \frac{\partial V_y}{\partial z} = -\frac{g}{\varrho} \cdot \left(\int_{z}^{zp(x,y)} \frac{\partial \varrho(x,z)}{\partial y} \cdot \partial z + \frac{\partial Zp(x,y)}{\partial y} \cdot \varrho(x,y,z) |_{z=Zp(x,y)} \right) + \nu \cdot \Delta V_y \\ V_z |_{z=Z_p} = -\int_{Z_z}^{Z_p} \left(\frac{\partial V_z}{\partial x} + \frac{\partial V_y}{\partial y} \right) \partial z \\ \frac{\partial T}{\partial t} + V_x \cdot \frac{\partial T}{\partial x} + V_y \cdot \frac{\partial T}{\partial y} + V_z \cdot \frac{\partial T}{\partial z} = a \cdot (\Delta T) \\ \varrho = \Phi(T) \\ OBJ = \int \left(\left(V_z |_{z=Z_p} \right)^2 - \min \right) \end{cases}$$
(8)

Where OBJ is the objective function of in the optimization problem.

The integration in the equation determining the objective function is performed over the entire Z_p surface, and the optimization problem is solved for each time step in the evolution of the heat transfer process T in the

velocity field V_x , V_y , V_z . To verify the developed approach and demonstrate its ability to determine the convective transport of reserves, several test problems were solved.

The first test problem involved a pool of water, a weakly compressible liquid, subjected to non-uniform heating. The following conditions were assumed: the water surface on the right side of the basin was cooled to a temperature of 4 degrees Celsius (the temperature of maximum water density), while on the left side, the water surface was heated to 30 degrees Celsius. The spatial steps were set to 1 meter. The evolution equations of system (8) were approximated by explicit schemes, with the explicit scheme (Salokhiddinov et al., 2022a; Salokhiddinov et al., 2022b; Salokhiddinov et al., 2023) being employed due to its capability to fully execute conservation laws across all possible configurations of the velocity vector field. The establishment method was utilized to solve the problem, and at each step of the solution evolution, an optimization problem was solved to determine the equilibrium position of the free surface of the moving water.

The selection of the topic of heat transfer in liquids is justified by the fact that problems of heat transfer in air and water are sufficiently similar to justify a unified approach. The findings of the study indicate that hydrodynamic heat transfer models developed for water content systems can be verified and effectively applied to air content systems. The high degree of accuracy demonstrated by CFD models in predicting heat and mass transfer, coupled with the successful empirical validation and sensitivity analyses conducted, confirm their applicability in a range of scenarios. Consequently, the analogous solutions obtained from examining heat transfer in both air and water can be employed for model validation (Kondjoyan et al., 2006; Talukdar et al., 2008; Sadafi et al., 2015; Tingfen et al., 2019).

The second test problem pertains to the primary objective of this paper, namely, the modeling of the formation of thermal islands in an arbitrary and complex configuration of the landform surrounding the calculation area. The ensuing conditions are stipulated as follows: two adjacent structures (high and low) are considered, with both structures situated on the right receiving radiant energy from the sun. In lieu of meticulously tracking the precise position and algorithmization of boundary conditions on the surfaces of impermeable objects within the calculation domain, it is proposed to subtract from the right side of the horizontal component of the equation of motion the product of the velocity by a coefficient that is significantly greater than one. Theoretically, this would allow air flow to pass through obstacles while simultaneously experiencing a significant braking effect from the obstacles. When the coefficient is set to a value of 100, the airflow's progression through obstacles becomes virtually halted. This heuristic approach facilitates the incorporation of obstacles of the most intricate and impermeable nature into the calculation algorithm, thereby ensuring their comprehensive consideration.

The medium of movement is air; the solution has been carried out by analogy with the analysis of fluid motion in Problem 1. The initial average air temperature was assumed to be 30°C, and this value was held constant over

the entire area of study. Spatial steps were taken equal to 1 meter. The system of equations (8) was utilized to solve this rather complex hydrodynamic problem.

The computations and the construction of the plots were carried out using programs written by the authors in the GAMS (General Algebraic Modeling System) programming language.

RESULTS & DISCUSSION

Results of solving the test problem 1. In order to obtain the stationary solution to the initial problem, a total of 200 iterations were performed. The calculation grid comprised 36 nodes in the horizontal direction and 60 nodes in the vertical direction. The graphical representation of the solution to the problem is depicted in Fig. 1. This Fig. illustrates the anticipated circular motion of the water in the vertical plane (longitudinal section of the pool) resulting from the uneven heating of the free surface.

This number of iterations was sufficient to form a

stationary field of temperatures and density. Based on this stationary field of density, a stationary field of motion velocities was also formed.

Fig. 2 shows the profile of the free surface of the water in the pool. A salient feature pertains to the absence of wave processes on the free surface and the precise fulfillment of the water balance on any vertical plane within the circular motion region. The accuracy of the water balance is ensured by the condition that the vertical component of the water velocity at the height of the free surface is equal to zero.

As illustrated in Fig. 3, the horizontal velocity of water in relation to the vertical at the center of the basin is examined. It is observed that the flow of water in the pool from left to right and from right to left is identical. This exact equality is also noted for all verticals. This equality is observed at each step of the evolutionary search for velocity and temperature fields. The duration of the optimization at each evolutionary step is approximately one second for a given number of nodes in the computational domain.



Fig. 1: Circular motion generated in the vertical plane in the pool by uneven heating of the free water surface in the pool.

Fig. 2: Profile of the free surface of the water in the basin



It is noteworthy that the precise equality of the water balance, as demonstrated in the solution to system (7), is attainable only at the nascent stages of the solution's evolution in conventional evolutionary computations of the position of the free surface.

Results of the solution of the test problem 2. The ensuing discussion will focus on the results of the solution to problem 2, which pertains to the formation of thermal islands in an arbitrary and complex configuration of the relief shape surrounding the calculation area. The graphical results of the solution are presented in Fig. 4, which illustrates the velocity field of the air involved in convective mixing in the presence of two buildings in the computational domain.

It is assumed that the right side of the building faces the sun and is heated four degrees Celsius more than the left side of the building. The average temperature of the air mixture entering the computational domain is assumed to be 30 degrees Celsius. A strong updraft is clearly visible on the right side of the domain and near the eastern building. The airflow quickly rises along the side of the building facing the sun. In addition, the airflow is pressed against the surface of the building as much as possible. On the non-sunny side of the tall building, there is a slight decrease in airflow. Most likely, this is a compensatory flow from the updraft on the sunny side of the low building further to the left. The lower building on the left side of the calculation area did not develop strong ascending air currents. The height of the lower task was not high enough to generate significant updrafts.

The second test problem was solved under the condition of free passage of the air mass through its left and right edges. Calculations have shown that with such an arrangement of buildings there is a shift of the entire airflow to the right. This means that not only local circulation currents occur, but also a general transfer of air from left to right. The heat from sunlight emitted on the right side of a tall building, which rises upwards, leaves the computational domain mainly through its right edge. There is a horizontal absorption of the entire air mass surrounding both the low and high buildings. A numerical experiment has shown that skyscrapers are not only able to change air currents locally, but also to shape them at considerable distances.

It is imperative to acknowledge and underscore the efficacy of the heuristic approach in accounting for obstacles within the calculation zone. The air flow velocities calculated within the confines of these obstacles are found to be negligible. This outcome indicates that the impact of obstacles on airflow is not achieved through the construction of boundary conditions within the solution area. Instead, it is accomplished by assigning forces of resistance to the movement of the flow by buildings - obstacles.



Fig. 3: The horizontal water velocity diagram on the vertical at the center of the basin.



Fig. 4: The field of air velocities surrounding buildings that are heated by sunlight from the right side.

Fig. 5 illustrates the temperature field's deviation from the mean value. The calculation indicates that due to the heating of the right side of the buildings, the temperature has increased to an average of 32 degrees Celsius. The color scale, presented in five shades, is located in 20 percent increments from the minimum to the maximum temperature deviation from the average value (assumed to be 32 degrees). More intense colors correspond to higher temperatures. The findings offer a comprehensive representation of air mass microcirculation and heat transfer in urban environments, paving the way for largescale urban microclimate modeling. As illustrated in Fig. 5, the maximum temperatures are observed to be concentrated within the buildings, aligning with the results reported by other researchers (Minh and Nguyen, 2019; Minh & Shukurov, 2020).

Further analysis of both test problems reveals strong alignment with previous research on convective flows and heat transfer dynamics in various environmental conditions. For Problem 1, the observed absence of wave processes on

the free water surface, as shown in Fig. 2, corroborates findings from prior simulations and experimental setups analyzing stationary fluid dynamics with distinct thermal gradients. The precise equality of water balance demonstrated in our calculations validates the methodological approach and strengthens the reliability of the numerical models. The results of Problem 2, particularly the air velocity field patterns shown in Fig. 4 and temperature deviations illustrated in Fig. 5, demonstrate the occurrence of localized thermal island effects around heated structures. The observed distinct upward and lateral flows align with significant air circulation patterns influenced by urban morphology and heating intensities (Fan et al., 2019). Our findings regarding thermal buildup on the non-sunny sides of buildings echo previous research showing that heat dissipation efficiencies vary depending on the aspect ratio and geometric configuration of urban structures.

The thermal deviations from average values observed in our second test problem are consistent with heat flux studies by Kahsay et al. (2019), which highlight how

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ji (0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.26	0.25	0.27	0.27	0.26	0.24	0.23	0.20	0.18	0.17			0.16	0.18	0.20	0.19		0.15	0.13		0.16	0.23	0.67	1.21	860	072	055	0.44		032	0.25	0.3	0.23	0.21	0.19	0.19
3	0.47	0.47	0.43	0.43	0.47	0.45	0.42	038	034	032	032	031	031	033	037	035	032	0.29	0.26	0.27	032	0.65	0.98	1.45	135	1.13	093	077	0.67	059		0.43	0.44	0.40	038	0.33
4	0.54	0.54	0.64	064	0.63	0.60	057	052	0.43	0.45	0.46	0.45	0.44	0.43	051	0.49	0.45	0.42	0.40	0.42	0.49	0.67	1.72	164	157	138	1.10	102	090	081	073	0.57	0.62	055	056	056
r r	0.74	0.74	0.76	0.76	0.75	0.22	0.62	044	0.50	0.57	057	0.57	0.57	0.61	045	0.67	050	0.55	0.52	0.00	0.42	040	1.46	121	1.72	126	1.22	1.71	1.00	0.00	0.01	0.05	0.75	0.75	0.77	0.72
P	070	070	0.10	0.70	075	072	0.08	004	0.29	0.57	037	031	0.07	0.71	0.05	005	0.39	033	035	0.20	0.00	0.90	1.00	101	175	1.70	1.01	1.21	1.77	0.99	0.91	400	0/3	0.75	073	073
<u>p</u>	Uas	0.85	0.85	Uas	Uas	Uat	011	073	0.59	0.67	0.67	0.64	0.09	0/4	0.78	076	072	0.66	0.68	0/5	026	1.13	1.87	198	189	1/1	155	13/	1.0	1.15	1.07	100	095	091	0.89	0.89
1	092	092	092	092	090	038	084	020	077	0.76	0.77	0.78	0.81	027	091	029	0.85	081	083	092	109	135	187	212	2/08	124	1.66	151	1.56	1.20	1.21	1.15	1.10	1.06	1.05	105
<u>p</u>	097	097	860	0.97	096	093	090	027	034	0.84	087	029	0.93	1.00	1.04	1.02	097	0.95	098	1.10	130	157	204	2.73	214	195	1.78	163	151	1.42	135	1.29	1.24	1.21	1.20	1.20
₽	1.02	1.02	1.02	1.02	1.00	098	0.96	093	091	0.92	096	099	1.04	1.12	1.17	1.14	1.10	1.09	1.14	1.29	150	177	2.18	232	2.22	205	1.88	174	1.63	155	1.48	1.42	1.39	137	136	136
j10	1.05	1.05	1.05	1.05	1.05	1.05	1.01	099	860	1.00	1.05	1.10	1.16	1.25	130	1.27	1.23	1.2	130	1.47	170	195	230	239	229	214	198	135	175	1.67	1.61	156	153	152	153	153
(11	1.10	1.10	1.10	1.10	1.09	1.05	1.05	1.05	1.05	1.08	1.15	1.21	1.27	137	1.42	139	136	137	1.46	165	123	211	239	2.44	235	2.21	207	195	1.85	1.79	174	170	1.69	1.68	170	170
82	1.13	1.13	1.14	1.14	1.13	112	1.11	1.11	117	1.17	1.74	131	139	1.49	153	151	1.43	151	162	182	205	226	2.47	2.47	240	23	216	205	1.97	191	187	135	184	1.85	187	187
33	1.17	1.17	1.17	1 12	1.17	1.17	1.17	1.17	1.10	1.8	174	1.47	150	1.60	145	167	1.60	164	177	100	278	240	253	250	744	776	7.74	215	208	204	201	200	200	202	204	204
2.4	1.70	1.70	1.74	1.74	1.77	4.33	1.77	1.72	1.76	1.22	1.03	183	1.61	1.74	175	1.72	1.77	1.70	107	2.00	332	262	750	350	2.0	241	199	37	2.00	216	244	214	216	312	3.33	2.22
<u>µ</u> •	1.00	1.20	1.21	1.21	1.22	1.22	1.22	1.0	1.00	1.35	1.60	152	101	1.02	1/2	1/5	172	1/0	195	215	2.57	494	2.59	200	2.07	2.41	2.00	2.0	2.00	2.00	244	249	2.10	210	2.22	2.26
<u>ps</u>	1.4	1.25	1.0	1.0	1.40	1.40	1.2/	1.0	1.55	1.42	155	162	1/2	182	185	185	136	191	207	2.00	2.65	260	264	252	2.69	2.00	2.00	235	251	2.8	2.05	2.8	252	255	2.59	2.29
j16	1.27	1.27	1.28	1.29	130	131	132	135	1.40	150	1.63	173	182	192	195	194	195	2.04	2.20	2.40	257	2.66	267	255	2.49	250	2/6	2,44	242	241	2.42	2.44	2.47	252	256	256
j17	130	130	131	133	134	135	137	1.41	1.43	1.59	172	182	192	201	205	204	2.06	216	231	250	264	270	2.68	258	2.49	255	255	254	253	253	255	258	2.63	267	272	272
j18	134	134	135	136	138	1.40	1.6	1.48	156	1.68	181	192	201	210	214	213	217	225	2.41	258	2.69	273	2.69	260	2.49	259	262	263	263	265	268	272	277	282	287	287
j19	137	137	1.39	1.40	1.42	1.45	1.49	155	1.64	1.76	190	200	210	218	2.22	2.22	2.26	236	250	264	272	274	268	261	253	261	268	271	274	277	281	286	291	297	3.02	3.02
120	1.41	1.41	1.42	1.44	1.47	150	155	162	171	184	198	209	2.18	226	230	230	235	244	257	269	274	274	267	262	257	260	273	279	283	288	292	298	3.04	3.10	3.15	3.15
121	1.44	1.44	1.45	1.48	151	155	1.61	1.68	178	192	206	217	2.26	234	237	238	2.42	251	263	272	275	270	2.65	262	261	262	278	226	292	298	3.08	3.10	3.16	3.22	3.28	3.28
722	1.47	1.47	1.49	152	156	160	1.66	175	1.85	200	214	2.24	233	2.40	244	245	2.69	258	267	274	275	269	264	262	264	268	283	293	3.01	3.07	314	3.20	3.77	334	339	3.39
78	151	151	152	156	160	165	177	121	107	207	2.21	321	240	7.65	250	251	255	263	271	276	278	260	262	761	267	274	785	700	3.08	3 16	3.78	2.50	2 27	3.85	350	250
24	101	1.54	1.84	140	144	1.70	1.70	1.07	1.00	214	177	120	2.40	363	100	254	260	367	372	374	374	369	200	260	360	1.10	200	204	215	2.72	2.24	2.10	2.47	2.00	240	7.60
1.0	100	100	100	100	1.00	170	1/0	107	200	2.20	2.0	2.00	2.00	204	200	2.00	200	2.07	270	270	223	200	200	200	230	200	200	3/0	2.20	2.20	3 30	2.00	2.00	200	3.00	240
9	15/	15/	1.00	1.04	109	1/5	185	195	200	2.00	239	245	251	251	209	201	200	2/0	2/5	2/0	4/4	2000	200	200	2/0	202	295	210	3.40	3.0	3.38	2.8/	320	3.04	3.09	2.09
120	160	150	1,63	1.67	173	130	189	199	212	2.20	239	2.49	256	261	263	265	258	273	276	275	2.69	261	251	257	271	285	299	312	3.25	335	3.45	355	3.65	372	3.76	376
μ <i>α</i>	1.63	163	1.66	171	177	134	194	205	2.18	232	2.65	253	2.60	264	266	268	271	274	275	272	2.65	257	2.47	255	272	287	3.02	3.17	330	3.41	352	3.63	372	3.79	382	382
jZB	1.65	1.65	1.68	174	120	139	198	210	2.23	237	2.49	257	263	267	268	270	272	275	274	269	261	252	2.42	252	273	290	3.05	3.21	335	3.47	359	3.69	3.78	384	387	387
<u>j</u> 29	167	167	171	1.76	184	193	2.08	215	2.28	2.42	253	261	2.66	2.69	270	271	273	275	273	266	256	2.46	236	250	273	293	3.08	3.Z	3.40	353	3.65	3.75	3.83	3.29	392	392
3 0	1.69	1.69	1.73	1.79	187	196	2.07	219	232	2.45	256	263	267	2.69	270	272	274	274	270	262	250	2.40	230	2.47	274	295	3.12	3.29	3.44	358	3.70	320	3.88	3.93	3.95	395
81	171	171	175	181	190	200	211	2.23	236	2.48	258	264	268	2.69	270	272	274	273	268	257	2.44	233	2.23	2.44	275	299	3.17	334	3.49	3.63	3.75	385	392	3.97	3.99	399
82	172	172	1.76	183	197	203	214	2.26	239	250	259	264	267	268	269	272	273	271	264	252	237	28	214	240	277	3.02	3.21	3 38	354	3.68	380	3.89	3.95	400	401	4.01
23	177	177	1.77	125	108	205	217	7.70	741	251	752	767	765	765	267	271	272	760	261	246	780	217	205	232	278	3.05	3.26	3.43	2.60	3.72	224	202	3.00	4.08	4.04	4.04
24	1.72	1.72	1.72	126	106	200	2.0	320	241	250	256	750	260	260	262	760	371	262	202	240	378	208	104	323	270	2.00	2 20	1/2	264	2.72	7 20	7.67	402	105	106	4.06
20	175	1.75	1.70	100	100	2.07	2.00	200	2.44	2.00	200	200	200	200	200	2.09	2/1	2.00	200	2.00	210	100	1.00	2.52	2/19	3.00	3.30	2.40	200	3/0	203	201	100	4.00	4.00	4100
p 5	1/2	1/2	1//	100	190	207	219	230	2.40	2.65	251	255	256	255	258	2.60	2.09	2.67	256	233	21/	199	182	2.0	2/19	3.12	3.35	300	3.09	382	392	400	4,00	6.07	600	61,0
<u></u> 26	171	171	176	185	195	207	219	2.29	238	2.43	244	244	2.44	2.63	250	261	255	264	252	227	210	129	170	222	278	3.14	3.38	357	373	326	3.96	4.03	4,07	4.09	4.09	409
 <u></u>	163	1.63	174	183	194	2.06	217	227	233	235	234	232	231	230	239	253	261	2.60	2.47	2.21	201	1.79	159	2.18	279	3.17	3.42	3.62	378	390	3.99	4.05	4.09	4.11	411	4.11
7 8	1.65	1.65	171	181	192	2.03	214	2.22	2.27	2.25	2.20	217	214	214	225	2.43	254	253	2.40	213	191	1.67	1.47	215	280	3.21	3.47	3.67	382	394	4.02	408	4.11	4.12	4.12	412
 <u></u>	161	151	1.67	177	188	199	2.09	215	217	212	208	198	194	193	207	231	2.45	245	231	208	179	152	1.33	211	282	3.Z	352	371	3.86	397	4.05	4.10	4.13	4.13	4.13	4.13
j40	156	156	1.62	172	183	193	202	206	2.06	197	184	175	170	1.70	126	215	233	234	219	190	154	136	1.18	2.09	285	3.29	357	376	390	4.00	4.08	412	4.14	4.14	414	414
j41	1.49	1.49	155	1.65	176	126	193	195	191	1.78	161	150	1.42	1.42	1.62	196	217	2.20	204	174	1.6	1.16	1.00	206	289	334	3.62	380	394	4.08	4.10	414	4.15	4.15	4.15	4.15
j42	1.42	1.42	1.48	157	167	1.76	182	182	174	156	134	1.21	114	1.12	136	174	199	2.02	185	152	1.21	092	081	205	293	339	3.67	385	398	4.06	412	4.15	4.16	4.16	4.15	4.15
j43	133	133	139	1.48	157	1.65	1.68	1.66	154	1.29	1.04	039	020	0.79	1.07	1.49	177	181	1.60	1.24	0.93	0.65	058	204	258	3.45	372	3 89	4.01	4.09	414	4.16	4.17	4.16	4.16	4.16
144	1.24	1.24	1.29	137	1.45	152	153	1.46	1.78	0.92	067	055	0.43	0.42	030	1.74	153	154	1.78	0.85	0.53		032	205	3.04	351	3.77	3.93	404	411	415	4.17	4.17	4.17	416	416
105	1.14	1.14	1.12	18	137	1 37	1 75	178	0.94	0.71	0.20	0.72	0.00	0.00	056	1.01	1.78	1.35	097	0.70	0.20	0.00	0.01	2/8	310	357	387	3.07	4.07	413	416	418	418	4.17	416	4.16
146	1.00	1.00	1.00	1.02	1.12	1.76	1.12	1/0	0.72	0.20	0.30	0.12	0.00	0.00	0.42	0.21	1.05	1.00	020	0.30	0.00	0.00	0.01	244	2.12	262	2.97	101	410	110	4.17	4.10	4.10	4.17	4.40	4.10
147	100	1100	6.64	1.00	1.10	1.41	1.00	100	075	0.20	0.20	015	0.00	0.00	0.0	041	110	1.02	0.70	0.0	0.0	0.00	0.01	24	2.10	202	247	401	4.10	9.42	4.17	4.10	4.10	4.47	4.12	4.15
14/	093	093	0.96	1.00	104	105	100	045	0.59	0.0	0.20	0.11	000	0.00	034	0.86	0.85	082	057	0.0	0.0	0.00	001	20	3.0	5.69	592	61,5	4.12	4,10	6.18	4.18	4.18	6.15	6.15	615
j43	082	082	0.85	0.88	091	091	025	071	0.49	0.20	0.20	010	0.00	0.00	0.28	053	0.59	0.66	0.47	0.20	0.20	0.00	0.01	227	332	372	396	408	414	4,17	4.18	4.18	4,17	4.16	414	414
j49	0.73	073	0.75	0.77	0.79	0.78	072	060	0.42	0.20	0.20	0.10	0.00	0.00	0.2	0.43	055	054	039	0.20	0.20	0.00	0.01	232	3.39	381	4.01	410	4.15	4.18	4.18	4.18	4.16	4.15	4.13	4.13
<u></u> 50	0.65	0.65	0.66	0.68	0.68	0.66	0.61	051	037	0.20	0.20	0.10	0.00	0.00	0.19	035	0.45	0.44	034	0.20	0.20		0.01	237	3.6	387	4.04	412	4.16	4.18	4.18	4.17	4.15	4.14	412	412
β 1	058	058	059	059	059	057	052	0.44	033	0.20	0.20		0.00	0.00	0.16	030	037	037	030	0.20	0.20	0.00	0.01	2.41	3.49	390	4.06	413	4.16	4.17	4.17	4.16	4.14	4.13	4.11	411
β 2	053	053	053	053	052	050	0.45	039	030	0.20	0.20		0.00	0.00	0.15	0.26	032	032	0.27	0.20	0.20	0.00	0.01	2.43	352	3.93	4.08	413	4.16	4,16	4.16	4.15	4.13	4.11	410	410
63	0.43	0.43	0.48	0.48	0.47	0.45	0.41	035	0.28	0.20	0.20	0.10	0.00	0.00	0.15	0.25	0.29	0.29	0.25	0.20	0.20	0.00	0.01	2.44	353	3.93	4.08	413	4.15	4.15	414	4.13	4.12	4.10	4.09	4.09
84	0.45	0.45	0.45	0.45	0.44	0.42	038	033	0.77	0.20	0.20	0.10	000	000	0.17	0.76	030	0.29	0.75	0.20	0.20	0.00	0.01	243	352	397	406	411	413	413	413	417	411	4.09	408	4/6
85	0.43	0.43	0.0	0.44	0.47	0.41	0.38	033	0.77	0.20	0.20	0.10	0.00	0.00	0.21	032	0.34	033	0.78	0.70	0.20	0.00	0.01	739	3.43	3.89	403	409	411	417	417	411	410	4.09	408	408
86	0.41	0.41	0.12	0.12	0.45	0.47	0.40	0.15	0.7	0.72	0.72	0.10	0.00	0.00	0.10	0.5	0.02	0.02	0.77	0.70	0.70	0.00	0.01	225	7.41	7.94	400	106	4.09	4.11	411	411	410	4.00	108	100
50	0.10	0.41	0.44	0.00	0.84	0.0	0.00	030	0.00	0.00	0.20	010	0.00	0.00	0.50	0.00	0.40	0.00	035	0.00	0.00	0.00	001	2.51	2,41	204	2.00	410	41.0	4.11	441	443	4.00	4.00	4.0	41.0
<u>67</u>	0.56	0.30	0.40	0.63	0.46	Q.Ar	0.66	0.42	033	0.20	0.0	010	000	000	0.6	0.57	071	0.61	0.62	0.0	0.0	000	001	219	331	3.78	596	404	4.05	4.10	4.12	412	4.11	6.11	4.10	4.10
58	032	032	037	0.63	0.49	054	057	054	0.63	0.20	0.20	0.09	0.00	0.00	072	106	1.12	0.95	054	0.20	0.20	0.00	001	196	3.21	3.75	3.95	4.05	4.09	4.12	414	414	4.14	4.13	4.13	4.13
<u>59</u>	0.22	0.22	030	0.42	053	064	0.73	0.77	83.0	0.20	0.20	0.07	0.00	0.00	1.2	170	120	158	0.61	0.20	0.20	0.00	0.01	151	330	384	4.03	410	414	4.16	4,17	4.18	4.18	4.17	4.17	417
<i>j</i> 60	0.02	0.02	0.19	038	057	0.76	0.95	1.14	133	0.20	0.20	0.00	0.00	0.00	2.48	265	283	3.05	0.10	0.20	0.20	0.00	0.00	0.10	4.22	4.22	4.22	4.22	4.22	4.22	4.22	4.22	4.22	4.22	4.22	4.22

Fig. 5: Field of air temperature deviations from the mean value due to convective transport caused by uneven heating of the buildings by the sunlight.

uneven heating leads to nuanced variations in convective .heat transfer coefficients (CHTC) across facade surfaces. This is particularly evident in our observations of upward flow dynamics near heated regions, underscoring the significance of vertical temperature gradients in driving localized microcirculations. The findings of Anbarsooz et al. (2022) provide further support for these observations, particularly in their analysis of heat transfer coefficients at building facades in different configurations. Our implementation of the heuristic approach to describing obstacles in airflow, combined with the optimization method, addresses some of the uncertainties discussed by Mirsadeghi et al. (2013) regarding external heat transfer coefficient modeling. While our study utilizes reliable numerical approximations, we acknowledge that external variables, such as building geometry and environmental factors, can introduce variances in urban thermal modeling. This understanding is crucial for the accurate interpretation and application of our results.

The practical implications of our findings for urban planning and design are significant, particularly in the context of the observed thermal island formation and air circulation patterns. These findings align with Battista (2017), who advocate for the integration of computational models in planning processes to enhance the prediction of heat flow dynamics in urban canyons. The work of Vollaro et al. (2015) further supports our approach, particularly their CFD analysis of convective heat transfer coefficients on external building surfaces. The successful application of our optimization-based method and heuristic approach to obstacle representation demonstrates the potential for efficient modeling of complex urban thermal dynamics.

Conclusion

Problems of this type are usually solved approximately by finding a stationary solution in evolutionary equations. This method requires a very large number of iterations and contains manifestations of wave processes in the solutions. Strong restrictions on time steps in iterations appear and the solution process takes a significant amount of time. The proposed method for the approximate solution of hydrodynamic and aerodynamic problems, consisting in the combination of the evolutionary approach and the optimization of a number of variables in the process of their search at each iterative step, allowed: to obtain the solution quickly and to achieve the exact fulfillment of the laws of conservation of mass of the moving medium and the complete absence of wave phenomena, which are not of interest for this type of problems. A new finite difference scheme (Salokhiddinov et al., 2022a; Salokhiddinov et al., 2022b; Salokhiddinov et al., 2023) was used to determine the transport of matter, momentum and heat energy in solving the problem. This scheme ensures the exact fulfillment of the conservation laws of matter, momentum and thermal energy for any configuration of velocity fields.

A heuristic approach to describing obstacles in the path of air flow has been tested and shown to be effective. This study contributes to the understanding of convective heat transfer in urban settings by providing both methodological advances and practical insights. The combination of our optimization approach with the heuristic treatment of obstacles offers an efficient way to model complex urban geometries while maintaining solution accuracy. These findings not only validate existing research but also provide new insights into the complex dynamics of urban heat transfer, particularly in the context of modern architectural configurations and urban densification. Subsequent research endeavors should prioritize the extensive validation of the obtained results on the scale of high-rise city development, leveraging micrometeorological stations. The findings will be instrumental in formulating measures aimed at enhancing the thermal comfort of urban streets.

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